



**ANNUAL Meeting 2008** 

August 6, 2008

## Transient mold fluid flow with well- and mountainbottom nozzles in continuous casting of steel

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Presented by Rajneesh Chaudhary



## Project Overview

- Well- and mountain-bottom nozzles have been investigated experimentally as well as numerically.
- Experiments have been performed to measure velocity below SEN using PIV in the mold of well-bottom nozzle. Impeller velocity probe was used to measure jet and surface velocities in the molds of both nozzles.
- The time average velocities, turbulence kinetic energies, frequency distribution of fluctuations, and power spectrums have been analyzed for both nozzles.
- A computational model has been formulated to solve 3-D, steady, incompressible Navier-Stokes equations with standard k-e model (RANS approach) using FLUENT.
- Model predictions and measurements are compared and combined to draw conclusions on the flow quality in the mold of both nozzles.
  Furthermore, predictions from water model were compared with full scale steel caster simulations.



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### The geometry of 1/3<sup>rd</sup> water model with well-bottom nozzle



Process parameters of 1/3<sup>rd</sup> water model and



### corresponding full-scale steel caster



Impeller velocity probe locations and orientations



- 1) Probe: 35mm long tube, 22/28 mm inner/outer dia
- 2) Propeller rotating in proportion to flow speed.
- Total response time of probe is ~10 sec, (electronic (~0.4 s to reach 63%) and mechanical response time)
- To measure jet velocity, probe is aligned with port angle (25 degree downward) at the bottom of the port.

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## Isometric view of well-bottom nozzle and strand meshes





Discussion on PIV measurements and simulations nuous asting Shadow spoils PIV measurements on the right side below SEN 1<sup>st</sup> order upwind convection scheme gives iet thickness and profile matching closely with PIV measurements 2<sup>nd</sup> order upwind convection scheme is more consistent with flow patterns and streamlines However, 2nd order scheme has stability issues with thinner jet (especially for jet coming from mountain-bottom nozzle) Therefore, 1<sup>st</sup> order upwind-scheme was used in further simulations.



# Comparison of simulations with measurements (average velocity and turbulent kinetic energy)

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	Jet Velocity (m/s)			Turbulent kinetic energy(m <sup>2</sup> /s <sup>2</sup> ) (x 10 <sup>-3</sup> )		
	Water model		Fluent	Water model		Fluent
	Left side	Right side		Left side	Right side	
Well bottom	0.687	0.685	0.69	0.0611	0.0898	22.3
Mountain bottom	0.957	0.944	0.92	0.0216	0.0087	20.1

#### Surface velocity

	Horizontal Surface Velocity (m/s)			Turbulent kinetic energy(m <sup>2</sup> /s <sup>2</sup> ) (x 10 <sup>-3</sup> )		
	Water model		Fluent	Water model		Fluent
	Left side	Right side		Left side	Right side	
Well bottom	0.103	0.115	0.11	0.31	0.38	1.4
Mountain bottom	0.148	0.166	0.18	2.23	3.14	2.4

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Compare simulations with measurements (average velocity)

- Well-bottom nozzle velocity measurements show little variation between sides, and agree with the predictions within 1%.
- Mountain-bottom nozzle exhibits significant asymmetry between left and right, indicating short time averaging.
- Predictions agree within these variations. For example, surface velocity averaged over the last 500s (0.180m/s) matches exactly with the prediction.

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- Agreement with the turbulent kinetic energy measurements is not quite as good except at the surface.
- Impeller probe is unable to respond to the high-frequency jet turbulence fluctuations due to the inertia of the impeller. Moreover, fixed probe orientation misses non-axial fluctuations.
- Measurements and predictions agree reasonably at the surface for the mountainbottom nozzle. (being low frequency)





## Calculated jet characteristics in both nozzles

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Weighted Average Parameter	Well bottom nozzle	Mountain type nozzle			
Port x-velocity (outward) (m/s)	0.75	0.92			
Port y-velocity (downward) (m/s)	0.48	0.52			
Port z-velocity (horizontal) (m/s)	0.065	0.076			
Port turbulent kinetic energy (m <sup>2</sup> /s <sup>2</sup> )	0.040	0.018			
Port turbulent dissipation rate (m <sup>2</sup> /s <sup>3</sup> )	2.11	0.64			
Vertical jet downward angle (deg)	32.8	29.3			
Horizontal jet angle (deg)	0	0			
Horizontal spread (half) angle (deg)	5.0	4.7			
Average jet speed (m/s)	0.89	1.06			
Back-flow zone	27%	30%			

Table- Jet characteristics in both nozzles

#### **Equations from**

H. Bai and B.G. Thomas, Turbulent Flow of Liquid Steel and Argon Bubbles in Slide-Gate Tundish Nozzles: Part I. Model Development and Validation. Metallurgical and Materials Transactions B, 2001. 32(2): p. 253-267. R Chaudhary

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# Discussion on jet and port flow qualities with both nozzles

- The jet in the well-bottom nozzle is more diffusive and thicker with a smaller back flow zone (27% vs. 30% in mountain bottom).
- In the mountain-bottom nozzle, flow goes straight along the side of the mountain with high velocity, producing a thinner and less diffusive jet with smaller horizontal spread- and vertical jet- angles.
- Secondary flows from the mountain bottom nozzle are weaker, as flow is directed more towards the narrow face.
- Higher outward, downward and horizontal weighted-average jet velocities exiting the mountain bottom nozzle are observed in both the experiments and computations.
- Turbulent kinetic energy is much higher in the well-bottom nozzle, with higher frequency fluctuations causing a more dissipative jet.





# Discussion on mold flow with well-bottom nozzle

- The higher dissipation rate leaving the port of the well-bottom nozzle causes the jet turbulent kinetic energy to decrease more as it moves through the mold.
- This thicker and more diffusive jet thus loses its momentum faster as it splits into upper and lower recirculation zones with weaker flow along the narrow face.
- Maximum velocity is found near the bottom of port exit, and is 1.23 m/s with the well-bottom nozzle.

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## Vertical velocity 10 mm from narrow face in both nozzles



1)The mountain-bottom nozzle has faster flow in the upper recirculation zone.

2) The jet impinges the narrow face at 180 mm below the top free surface with both nozzles.

3) Well-bottom nozzle has weak reverse flow at the bottom of the strand. However, strong downward flow is seen in mountainbottom nozzle.



- 1) Mountain bottom nozzle gives ~1.5 times higher horizontal surface velocity. This higher surface velocity agrees with measurements.
- 2) The mountain-bottom nozzle gives ~5 times higher turbulent kinetic energy compared to the well-bottom.
- This is due to the low frequency and high magnitude fluctuations in the surface velocity for this nozzle.

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Discussion on measured jet velocity



## Discussion on measured surface velocity



3) The general drop in energy observed with increasing frequency matches previous work. An exception is the small peak found at ~0.07 Hz (~14s).



Qualitative description of flow characteristics with well- and mountain-bottom nozzles

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	Jet velocity		Sur	face velocity	Asymmetry (Due to flow
	Average velocity	Fluctuations	Average velocity	Fluctuations	transients)
Well bottom	Low (Thick jet)	High (High frequency)	Low	Low (High frequency)	Low
Mountain bottom	High (Thin jet)	Low (Low frequency)	High	High (Low frequency)	High

## Qualitative description of the surface and jet velocities based upon simulations as well as experiments

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Differences between laboratory water model and steel caster

- Geometric scaling of all linear dimensions to 1/3rd
- No solidifying shell and stationary walls
- A domain bottom with water exiting through circular holes in a horizontal plate instead of a very long, graduallytapering flow domain
- Air above the free surface instead of powder, sintered and liquid slag layers.

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0.1

Horizontal s 0

Narror face

0.2 0.3 0.4 0.5 0.6 0.7 0.8 Normalized horizontal distance from narrow face to nozzle (distance/L)

SEN

0.9



Discussions on surface velocity in full scale caster (Cont...)

- Thus, the well-bottom nozzle is preferred over the mountain-bottom nozzle for this steel caster and conditions.
- If casting conditions produced very small surface velocities, then the mountain-bottom nozzle might appear to be better. However, the results of this work suggest that changing the flow pattern in some other way and using the well-bottom nozzle is the best solution.

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## Discussions on free surface level

- The free surface level without shell and air above matches most closely with water model, as expected, although the water model underpredicts by a factor of 2.3.
- Introducing the shell and adding slag both increase the profile variations.
- Thus, the water model underpredicts surface level variations in the caster using Froude scaling.

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## Summary



- This work investigates well-bottom and mountain-bottom type nozzles both experimentally and numerically.
- The computational model agrees very well with measured velocities in all cases, but overpredicts turbulent kinetic energy in the jet and surface of the well bottom nozzle perhaps due to time resolution (~0.1 Hz) of the impeller probe and fluctuations being higher frequency.
- The measured surface turbulence in mountain bottom nozzle matches well with the simulations.





- Laboratories, South Korea.
- Continuous Casting Consortium, University of Illinois at Urbana-Champaign, POSTECH, and POSCO, South Korea
- Seong-Yeon Kim and Graduate students at Metal Processing Simulation Laboratory, UIUC.
- Fluent and ANSYS Inc.